# Numerical simulation of fatigue crack growth

1st Winter School on Trends on Additive Manufacturing for Engineering Applications

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#### **Material fatigue**

- Fractures due to dynamic loads have been occurring in load-bearing structures since ancient time, but the study of fracture has intensified when metals became the dominant material in the construction of structures.
- During the 20th century, we learned that repeated loads lead to a process called *material fatigue*.
- Def: *Fatigue* is the weakening of a material caused by cyclic loading that results in progressive and localized structural damage and the growth of cracks.

#### Material fatigue example



#### Safe-life and fail-safe approach

- The conventional approach in design of the fatigueresistant structure is based on the evaluation of the weakest component's life. This approach is known as *safe-life* and does not take the crack growth into account. At the end of the safe operational life, the component is automatically retired from service (landing gear, wing-fuselage attachment, engine mount, etc.)
- The *fail-safe concept*, on the other hand, is based on the argument that even if an individual member of a large structure fails, there should be sufficient structural integrity in the remaining parts (*structural redundancy*) to enable the structure to operate safely until the damage is detected and repaired (wings, fuselage, engine covers, etc.)

#### **Bad fail-safe design**



Hawaii, Aloha Flight 243, Boeing 737, an upper part of the plane's cabin area rips off in mid-flight. Metal fatigue was the cause of the failure.



#### Multiple crack growth in the fuselage skin (XFEM)



### **Different phases of the fatigue life**

• The fatigue life until failure consists of two periods: the crack initiation period and the crack growth period.



• The stress intensity factor (SIF) K is used for predictions on crack growth. The SIF is a measure of the singular stress term occurring near the tip.

Image taken from: Fatigue of Structures and Materials, J. Schijve (2009)

#### **Regions of the crack growth rate**



#### **Equations for region II**



**R** is the stress ratio,  $\Delta K$  is the stress intensity range, and **m** is the slope on a log – log scale. The value  $\gamma$  is a material constant. The value  $C_0$  is the intercept constant C for the case where R = 0.

• The Paris equation does not account for the stress ratio R.

#### **NASGRO** equation for region II

• 
$$\frac{da}{dN} = C \left[ \left( \frac{1-f}{1-R} \right) \Delta K \right]^m \frac{\left( 1 - \frac{\Delta K_{th}}{\Delta K} \right)^p}{\left( 1 - \frac{K_{max}}{K_C} \right)^q}$$
 NASGRO equation

where:

$$f = \frac{K_{op}}{K_{max}} = \begin{cases} max(R, A_0 + A_1R + A_2R^2 + A_3R^3), & R \ge 0\\ A_0 + A_1R, & -1 \le R < 0 \end{cases}$$

$$A_0 = (0.825 - 0.34\alpha + 0.05\alpha^2) \cdot \left[ cos\left(\frac{\pi}{2}\sigma_{max} - \sigma_0\right) \right]^{\frac{1}{\alpha}}$$

$$A_1 = 0(415 - 0.071\alpha) \cdot \frac{\sigma_{max}}{\sigma_0} \qquad A_2 = 1 - A_0 - A_1 - A_3 \qquad A_3 = 2A_0 + A_1 - 1$$

$$\Delta K_{th} = \frac{\Delta K_0 \sqrt{\frac{a}{a+a_0}}}{\left(\frac{1-f}{(1-A_0)\cdot(1-R)}\right)^{1+C_{th}R}} \quad \frac{K_C}{K_{IC}} = 1 + B_K e^{-\left(A_K \frac{t}{t_0}\right)^2} \quad t_0 = 2.5 \cdot \left(K_{IC} / \sigma_{ys}\right)^2$$

Coefficients C, p, q and n are empirically obtained.

#### **Test specimens for evaluation of K**



CT specimen (a) comparison to CCT specimen (b)

#### **Stress intensity factor calculation**



# Analitical solution for K<sub>I</sub>

The stress intensity factors  $K_I$  and  $K_{II}$  are expressed as:

$$K_{I} = \sigma' \sqrt{\pi a} F_{I}(a/W) \qquad \qquad K_{II} = \tau' \sqrt{\pi a} F_{II}(a/W)$$

where *a* is the crack length, *W* is the width of the component, and  $\sigma'$ ,  $\tau'$  are characteristic stresses in the component.



$$\sigma_{22} = \frac{\sigma_{22}^{\infty} r}{\sqrt{r_1 r_2}} \left[ \cos\left(\theta - \frac{\theta_1}{2} - \frac{\theta_2}{2}\right) + \frac{a^2}{r_1 r_2} \sin\theta \sin\frac{3(\theta_1 + \theta_2)}{2} \right] + \frac{\sigma_{12}^{\infty} r}{\sqrt{r_1 r_2}} \frac{a^2}{r_1 r_2} \sin\theta \cos\frac{3(\theta_1 + \theta_2)}{2}$$

$$= \frac{(1+\nu)\sigma_{22}^{\infty}\sqrt{r_{1}r_{2}}}{4E} \left[ 4(1-2\nu)\cos\frac{\theta_{1}+\theta_{2}}{2} - \frac{4r(1-\nu)}{\sqrt{r_{1}r_{2}}}\cos\theta - \frac{2r^{2}}{r_{1}r_{2}}(\cos\frac{\theta_{1}+\theta_{2}}{2}) + \cos\left(2\theta - \frac{\theta_{1}}{2} - \frac{\theta_{2}}{2}\right) \right] + \frac{(1+\nu)\sigma_{12}^{\infty}\sqrt{r_{1}r_{2}}}{E} \left[ 2(1-\nu)\sin\frac{\theta_{1}+\theta_{2}}{2} - \frac{2r(1-\nu)}{\sqrt{r_{1}r_{2}}}\sin\theta + \frac{r^{2}}{r_{1}r_{2}}\sin\cos\left(\theta - \frac{\theta_{1}}{2} - \frac{\theta_{2}}{2}\right) \right]$$

Source: Theo Fett, Stress Intensity Factors – T-Stresses – Weight Functions (2008)

#### Software for numerical methods

- NASGRO v4 (boundary element method)
- FRANC2D (finite element method)
- FRANC3D (finite element method)
- Ansys (finite element method)
- MSC Fatigue (finite element method)
- Abaqus (extended FEM)
- Code Aster (extended FEM)
- Zebulon (extended FEM).

https://www.swri.org/consortia/nasgro

# NASGRO v4 software (2D models)

THE C	Units	U.S. Units; inches, ksi.	ksi sart(in)	
	Tooltips	► ✓ S.I. Units: mm, MPa, N	/Pa sqrt(mm)	
T	Set user tool			
Т	Save options now			
145	Save options on exit			
25.4 Hole c  127	tr-to-edge dist, B defining dim	units selection		$S_{0} + \frac{r}{Wt}$ $S_{1} = \frac{6M}{Wt^{2}}$ $S_{3} = \frac{P}{Dt}$ $t = thickness$ $S_{0}$

#### **NASGRO v4 material selection**

Select Geometry	oose Material	K						
Select interaction model Non-Interaction Boeing Const. Closure Generalized Willenborg Chang Willenborg Strip Yield	Material property set 1 Select data source NASGRO material file User material file New data		Available da/dN-vs-dK data forma NASGRO equation constants ESA/NLR equation constants Walker equation constants			ats C 1-D table C 2-D table; same da/dN set for all R C 2-D table; diff da/dN set for each R		
	Category:	[M] 1000-9000 S	ERIES AL		-	Show faves	Enter ID	View curvefit
	Group:	[2] 2000 series			-	Add to faves	Reload mat	1 <u>da</u> 🧬
	Allow	[EA] 2024-T3 AI				-		dN
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#### **NASGRO v4 output files**



https://cfg.cornell.edu/software/

# FRANC2D/L (FME model)



#### NASGRO i FRANC2D/L comparison



### **Calculation of fatigue life**



https://franc3d.in/

#### FRANC3D (FME model)



# $K_{I}$ and $K_{II}$ calculated in FRANC3D



#### $K_I$ values calculated along the crack front



 $K_{II}$  values calculated along the crack front

#### **Equivalent SIFs and kink angle**

 For automated crack growth the following equations for *equivalent SIF are used*:

$$K_{eq} = \sqrt[4]{K_I^4 + 8K_{II}^4}$$

$$K_{eq} = \frac{1}{B} \sqrt{(k_1)^2 + \left(\frac{k_2}{s}\right)^2 + A(k_H)^2}$$
$$K_{eq} = \sqrt{K_I^2 + K_{II}^2}$$

• Kink angle formula:

$$= -\arccos\left[\frac{2K_{II}^{2} + K_{I}\sqrt{K_{I}^{2} + 8K_{II}^{2}}}{K_{I}^{2} + 9K_{II}^{2}}\right]$$

https://www.ansys.com/

# **Ansys SMART technology (FEM)**

- SMART: Separating, Morphing, Adaptive and Re-meshing Technology
- Calculates Mode I, II, III Stress Intensity Factors (SIFs)
- Supports static crack propagation based on failure criteria using SIFs or J-Integral
- Supports fatigue crack propagation based on Paris' law
- Supports crack propagation of internally generated crack meshes including semi-elliptical and arbitrary cracks
- Supports crack propagation of pre-meshed cracks
- Limited to isotropic linear elastic analyses (no plasticity, no nonlinear geometry effects, no load-compression effects, no cracktip-closure effects)
- Assemblies are supported, but only MPC (multi point constraint) formulation can be used (no frictional or frictionless contact)
- Supports multiple cracks in the model
- Thermal loads and imported loads (pressure) can be used.

### **Extended FEM (XFEM)**

- Enables the modeling of a discontinuous field independently of the generated finite element mesh.
- XFEM does not require mapping between the mesh and geometry of discontinuity .
- It is possible to use an arbitrary crack shape, and the fatigue crack growth simulation can be performed without generating new nodes around the tip as the crack grows.

https://edu.3ds.com/en/software/abaqus-student-edition

#### XFEM verification – Ex. 1 (TC03)



#### XFEM verification – Ex. 1 (TC03)



#### Ex. $1 - K_I$ values along crack front



# Ex. 1 : $K_I$ and $K_{eqv}$ values comparison



#### Ex. 1 – Stress change with growth



#### Ex. 1 – Displacement field



#### Ex. 2 – CT specimen SIF calculation





0,7

Dimensions and load of CT specimen

$$K_{I}^{(theor)} = \frac{P \cdot Y(c/w)}{B \cdot w^{1/2}} \qquad 0.3 \le \left(\frac{c}{w}\right) \le$$

$$Y(c/w) = 39.7 \left(\frac{c}{w}\right)^{\frac{1}{2}} - 294 \left(\frac{c}{w}\right)^{\frac{3}{2}} + 1118 \left(\frac{c}{w}\right)^{\frac{5}{2}} - 1842 \left(\frac{c}{w}\right)^{\frac{7}{2}} + 1159 \left(\frac{c}{w}\right)^{\frac{9}{2}}$$

 $K_{I}^{(theor)} = \frac{200N \cdot 10.17348}{1.25mm \cdot (30mm)^{1/2}} = 297,187MPamm^{0,5}$ 

#### Ex. 2 – CT BEM and XFEM results



Abaqus result (XFEM):  $K_I = 292,50 MPamm^{0,5}$ (difference 1.57%)

NASGRO v4 result (BEM):  $K_I = 282,024MPamm^{0.5}$ (difference 5.1%)



#### Ex. 2 – CT specimen FEM result



#### Ex. 2 – CT specimen stress (FEM)



#### Ex. 2 – CT specimen stress (XFEM)


## Ex. 2 – CT specimen (displacement)



## Ex. 3 – CCT specimen (stress field)



## Ex. 3 – CCT specimen (displacement)





### Ex. 4 – Non-standard specimen (stress field)



# Ex. 4 – non-standard specimen (displacement)





## Ex. 5 – Three-point flexural test



# **CASE STUDIES**

- Damaged wing-fuselage attachment
- Crack growth in the wing spar
- Fatigue life assessment of damaged integral skin-stringer panel

## CS 1: Damaged wing-fuselage attachment

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Aircraft fuselage stress analysis (thin-walled structure analysis in Femap and NX Nastran)

#### **CS 1 - Damaged wing-fuselage attachment**



(Femap and NX Nastran analysis)

#### CS 1 - Damaged wing-fuselage attachment



#### Wing-fuselage attachment



#### CS 1 - Crack growth in the lug (FRANC2D)



## CS 1 - Crack growth in the lug (Abaqus XFEM)



Initial penny-shaped crack in hexahedral mesh, and crack after 6th step of propagation



Initial penny-shaped crack in tetrahedral mesh, and crack after 6th step of propagation

## CS 1 - Crack growth in the lug (Abaqus XFEM)



Number of cycles vs. crack depth for hexahedral and tetrahedral mesh

## CS 1 - Through crack (XFEM vs FEM)



#### Initial through crack, and crack at the end of propagation



## CS 1 - Through crack (XFEM vs FEM)



Comparison of values

## CS 1 - Lug stress with growth (XFEM)



## CS 1 - Lug displacement (XFEM)



# CS 2: Crack growth in the wing spar

## CS 2 – Fatigue life of the wing spar (2024-T3)



## CS 2 – Experimental setup for fatigue testing



#### CS 2 – FEM identification of the critical area



## CS 2 – Cracks' initiation and growth



Cracks' growth paths on the vertical walls of the left and right spar cap

## CS2 – The spar after experiment



## **CS2 – FEM simulation in FRANC2D**



 $1^{st}$  and  $2^{nd}$  crack path obtained in FRANC2D



Cracks' paths observed in experiment



## CS 2 – Estimated fatigue life



Graph Crack length vs. Life obtained after integration of NASGRO formula

## CS 2 – XFEM calculation (Abaqus)



### CS 2 – Crack growth on horizontal cap wall



#### CS 2 – Crack growth on vertical cap wall



#### Crack shape after 22<sup>nd</sup> step of propagation

24<sup>th</sup> step of propagation



Crack shape after 45<sup>th</sup> step of propagation

## CS 2 – Comparison with experiment



The crack path on the spar cap in the experiment (left) and simulated crack path (right)

## CS 2 – Comparison with experiment



View of the crack on the cap (left) after stabilization (residual stress removal) and simulated crack path (right)

# CS 2 – Crack growth (displacement)







## CS 2 – Calculated SIF values (Abaqus)



Values of equivalent stress intensity factor  $(K_{eqv})$  as a function of crack length

## CS 2 – Fatigue crack growth life



## CS 2 – Comparison of results

	14 18 18		NY AV AV SA		
Ansys (crack occurrence)	FRANC2D (crack growth)		Total (2D)	Abaqus (crack growth)	Total (3D)
7944 cycles	1 <sup>st</sup> crack	40,413 cycles	48,357 cycles	50,743 cycles	58,687 cycles
	2 <sup>nd</sup> crack	13,392 cycles	21,336 cycles		

#### Fatigue life obtained using FEM and XFEM

Crack occurrence		Experiment stopped after	
1 <sup>st</sup>	8,452 cycles	58,520 cycles	
2 <sup>nd</sup>	39,450 cycles		

Fatigue life obtained in experiment

## CS 2 – Standard spectra in aircraft design


## CS 2 – Standard spectra and fatigue life



# CS 3: Fatigue life assessment of damaged integral skin-stringer panel

#### CS 3 – Integral skin–stringer panels



## CS 3 – Integral skin–stringer panels



#### CS 3 – Experimental analysis of panels









Model of 4-stringer plate (2mm mesh) with crack after 117 steps of propagation



Crack growth vs. number of cycles (m = 3.174 and C =  $1.77195 \times 10^{-12}$  MPa mm<sup>1/2</sup>)



Model of 4-stringer plate (4mm mesh) with crack after 278 steps of propagation



Crack growth vs. number of cycles (m = 3.174 and C =  $1.77195 \times 10^{-12}$  MPa mm<sup>1/2</sup>)



#### CS 3 – Comparison of values



The number of cycles to critical crack length obtained in Abaqus is still less than the number of cycles obtained in the experiment (290743 cycles versus 422328 cycles; difference of about 31 %).

## CS 3 – Improvements in numerical model



Mesh details of 4-stringer model. Weld line is presented.



Different definition of boundary conditions

# Thank you for your attention!